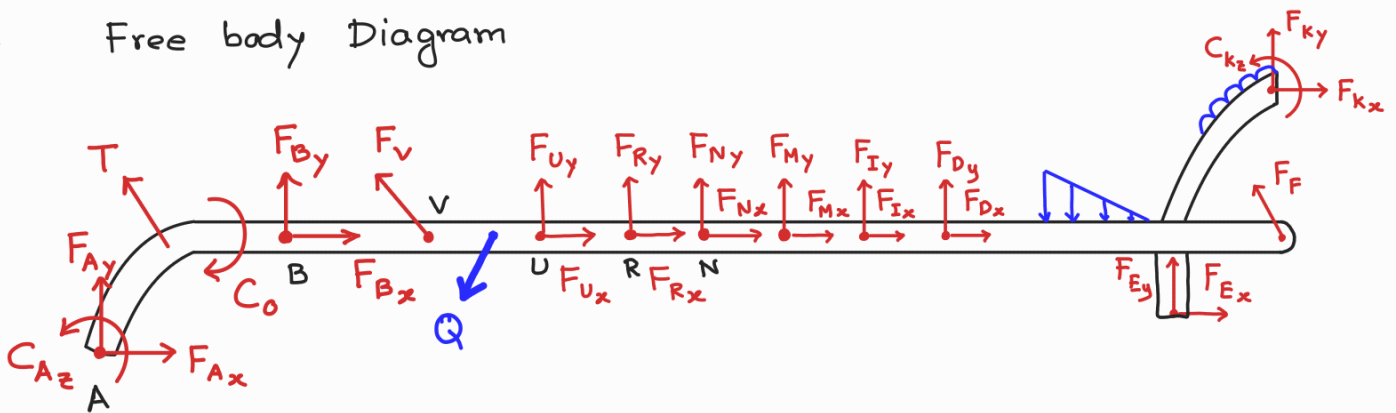


Part A

The diagram illustrates a mechanical system with the following components and forces:

- Applied torque:** Indicated by red curved arrows labeled C_0 and C_1 .
- Applied distributed force:** Indicated by a green arrow pointing to a triangular load on a curved member.
- Applied force:** Indicated by blue arrows labeled Q and T_1 .
- Other components and labels:**
 - Members:** Cable, Bar BC , Bar CD , Bar DE , Bar EF , Bar FG , Bar GH , Bar HI , Bar IK .
 - Supports:** Pin supports at B , C , D , E , F , G , H , I , J , K ; Roller supports at A , L , M , N , O , P , R , S , T , U , V , W , X , Y , Z , AA , BB , CC , DD , EE , FF , GG , HH , II , JJ , KK , LL , MM , NN , OO , PP , QQ , RR , SS , TT , UU , VV , WW , XX , YY , ZZ .
 - Other labels:** Q , T_1 , T_2 , T_3 , T_4 , T_5 , T_6 , T_7 , T_8 , T_9 , T_{10} , T_{11} , T_{12} , T_{13} , T_{14} , T_{15} , T_{16} , T_{17} , T_{18} , T_{19} , T_{20} , T_{21} , T_{22} , T_{23} , T_{24} , T_{25} , T_{26} , T_{27} , T_{28} , T_{29} , T_{30} , T_{31} , T_{32} , T_{33} , T_{34} , T_{35} , T_{36} , T_{37} , T_{38} , T_{39} , T_{40} , T_{41} , T_{42} , T_{43} , T_{44} , T_{45} , T_{46} , T_{47} , T_{48} , T_{49} , T_{50} , T_{51} , T_{52} , T_{53} , T_{54} , T_{55} , T_{56} , T_{57} , T_{58} , T_{59} , T_{60} , T_{61} , T_{62} , T_{63} , T_{64} , T_{65} , T_{66} , T_{67} , T_{68} , T_{69} , T_{70} , T_{71} , T_{72} , T_{73} , T_{74} , T_{75} , T_{76} , T_{77} , T_{78} , T_{79} , T_{80} , T_{81} , T_{82} , T_{83} , T_{84} , T_{85} , T_{86} , T_{87} , T_{88} , T_{89} , T_{90} , T_{91} , T_{92} , T_{93} , T_{94} , T_{95} , T_{96} , T_{97} , T_{98} , T_{99} , T_{100} .

Soln : Free body Diagram



T (tension) is along the cable

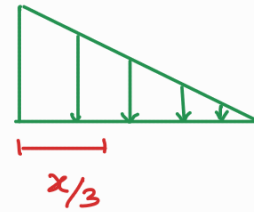
C_o is the moment due to an applied couple

B is a pin joint

F_v at point V is a slot connection

U, R, N, M, I, D are pin joints

The distributed force can be replaced by a single concentrated force at $x/3$ from left side



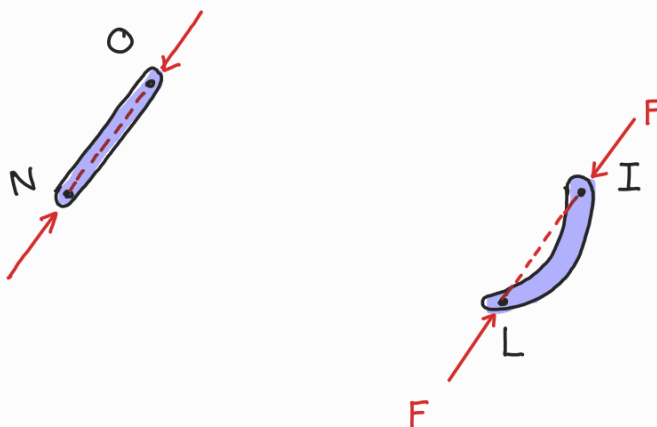
E is a 2D ball and socket joint, similar in functionality as a pin joint

F is a roller support. Reaction force is perpendicular to its direction of free translational movement

K is a fixed support: two forces and one moment

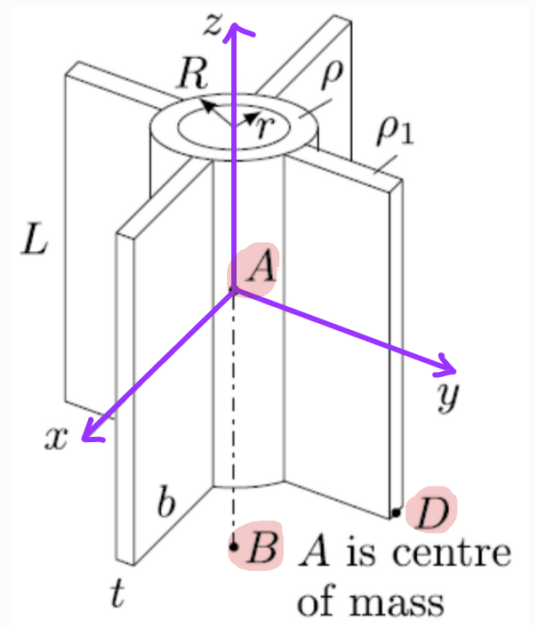
Note: The rod ABCDEFK is constrained and can't move it: it is stationary

Later, after studying statics, you will know that members 'NO' and 'IL' are "two-force" members



2) Find the inertia matrix $[\underline{I}^A]$ of the composite RB at A relative to csys shown.

Density of annulus is ρ and that of plates is ρ_1 .



Find principal axes of the composite RB at point A, B, and D.

Solu:

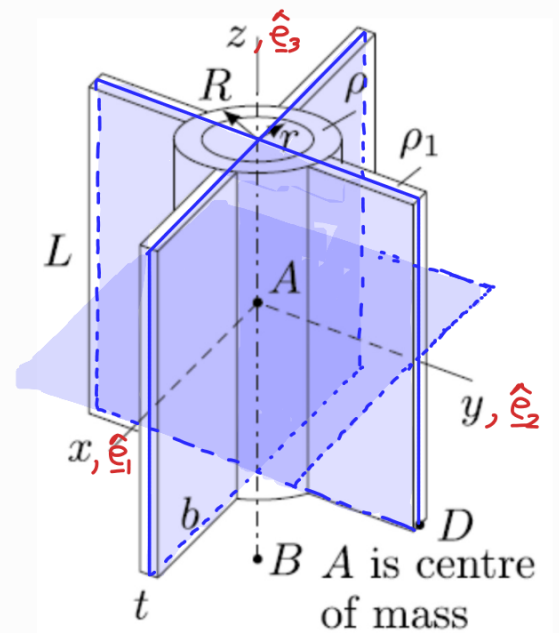
Inertia matrix at COM A

Visual inspection $\rightarrow$ Is this a body of revolution? NO
 $\rightarrow$ Does the body have planes of symmetry? YES

Three planes of symmetry through A

- ① xz -plane $\Rightarrow$ y -axis as p -axis
- ② yz -plane $\Rightarrow$ x -axis as p -axis
- ③ xy -plane $\Rightarrow$ z -axis as p -axis

$$\therefore [\underline{I}^A]_{\begin{pmatrix} \hat{e}_1 \\ \hat{e}_2 \\ \hat{e}_3 \end{pmatrix}} = \begin{bmatrix} I_{xx}^A & 0 & 0 \\ 0 & I_{yy}^A & 0 \\ 0 & 0 & I_{zz}^A \end{bmatrix}$$



Further by symmetry, $I_{xx}^A = I_{yy}^A$

So we need to find only two scalars I_{xx}^A and I_{zz}^A to completely define $[I^A]$

$B_1 \equiv$ Hollow Cylinder

$B_{2-5} \equiv$ Rectangular plates (not thin)

Calculation of $I_{xx}^A (= I_{yy}^A)$

$$I_{xx}^A = I_{xx}^A(B_1) + \sum_{i=2}^5 I_{xx}^A(B_i)$$

Hollow circular cylinder (Lec 12)

$$\text{Mass, } m = \rho L \pi (R^2 - r^2)$$

$$I_{xx}^A(B_1) = \frac{m}{4} (r^2 + R^2) + \frac{m L^2}{12}$$

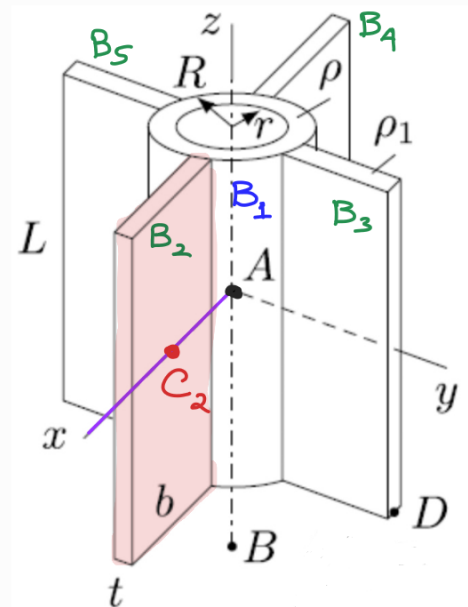
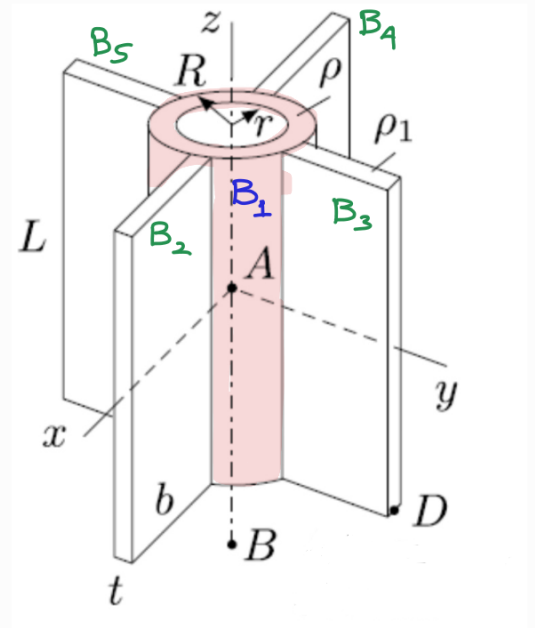
Rectangular plates (Lec 12)

$$\text{Mass, } m_1 = \rho_1 L t b$$

// axes thm

$$I_{xx}^A(B_2) = I_{xx}^{C_2}(B_2) + m_1 (\text{posi. of } C_2 \text{ from } A \perp \text{ to } x\text{-axis})^2$$

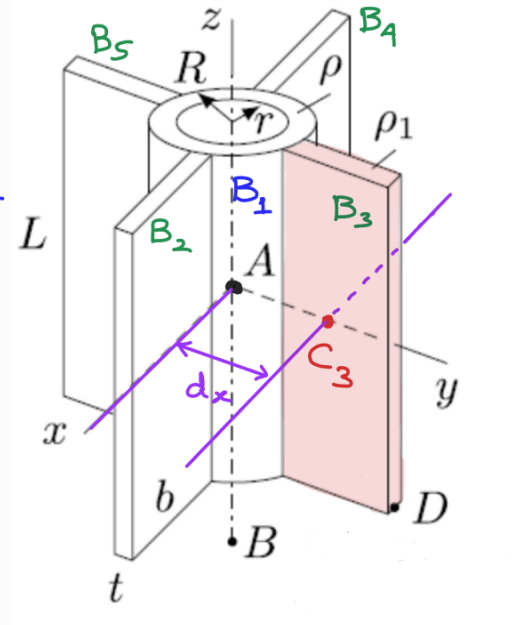
$$= \frac{m_1 (L^2 + t^2)}{12}$$



Note the $I_{xx}^A(B_4) = I_{xx}^A(B_2) = \frac{m_1(L^2 + t^2)}{12}$ ✓

// axes thm
 $I_{xx}^A(B_3) = I_{xx}^{C_3}(B_3) + m_1(\text{posi. of } C_3 \text{ from } A \perp \text{ to } x\text{-axis})^2$

$$= \frac{m_1(L^2 + b^2)}{12} + m_1\left(R + \frac{b}{2}\right)^2$$



Similarly, the $I_{xx}^A(B_5) = I_{xx}^A(B_3) = \frac{m_1(L^2 + t^2)}{12}$

$$I_{xx}^A = I_{xx}^A(B_1) + \sum_{i=2}^5 I_{xx}^A(B_i)$$

$$= \underbrace{\frac{m}{4}(r^2 + R^2) + \frac{mL^2}{12}} + \underbrace{2\left(\frac{m_1(L^2 + t^2)}{12}\right)} + \underbrace{2\left(\frac{m_1(L^2 + t^2)}{12}\right)} + \underbrace{2\left(m_1\left(R + \frac{b}{2}\right)^2\right)}$$

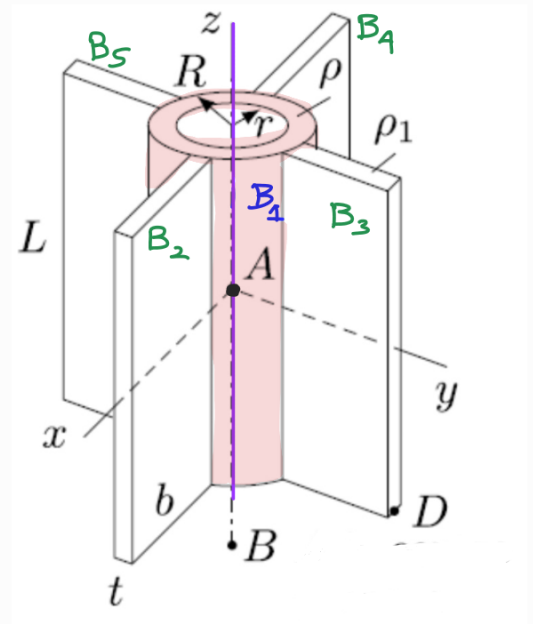
Calculation of I_{zz}^A

$$I_{zz}^A = I_{zz}^A(B_1) + \sum_{i=2}^5 I_{zz}^A(B_i)$$

Hollow circular cylinder (Lec 12)

Mass, $m = \rho L \pi(R^2 - r^2)$, $A \equiv \text{COM}$

$$I_{zz}^A(B_1) = \frac{m}{2}(r^2 + R^2) //$$

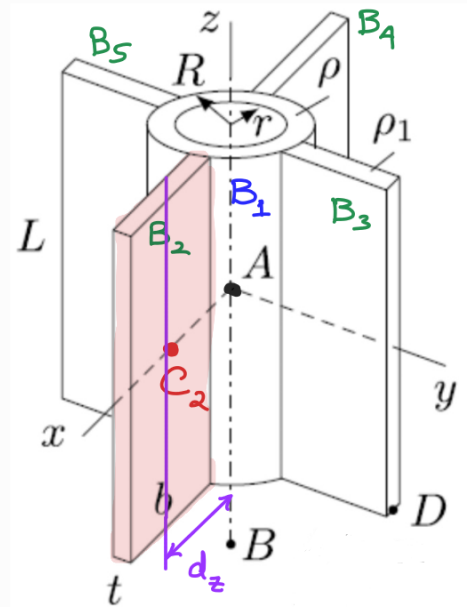


Rectangular plates (Lec 12)

Mass, $m_1 = \rho_1 L t b$

// axes thm
↓

$$\begin{aligned} I_{zz}^A(B_2) &= I_{zz}^{C_2}(B_2) + m_1 \left(\text{posi. of } C_2 \text{ from } A \perp \text{ to } z\text{-axis} \right)^2 \\ &= \frac{m_1(b^2 + t^2)}{12} + m_1 \left(R + \frac{b}{2} \right)^2 \end{aligned}$$



Recognize that I_{zz}^A of all four rectangular plates are SAME

$$\therefore I_{zz}^A(B_2) = I_{zz}^A(B_3) = I_{zz}^A(B_4) = I_{zz}^A(B_5) //$$

$$\begin{aligned} I_{zz}^A &= I_{zz}^A(B_1) + \sum_{i=2}^5 I_{zz}^A(B_i) // \\ &= \frac{m}{2} (r^2 + R^2) + 4 \left[\frac{m_1(b^2 + t^2)}{12} + m_1 \left(R + \frac{b}{2} \right)^2 \right] \end{aligned}$$

$$[\underline{I}^A] \begin{pmatrix} \hat{e}_1 \\ \hat{e}_2 \\ \hat{e}_3 \end{pmatrix} = \begin{bmatrix} I_{xx}^A & 0 & 0 \\ 0 & I_{yy}^A (= I_{zz}^A) & 0 \\ 0 & 0 & I_{zz}^A \end{bmatrix} \leftarrow \text{diagonal}$$

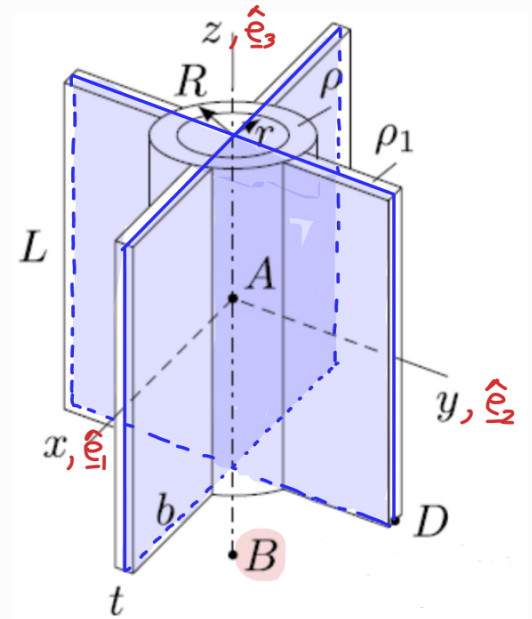
Since $[\underline{I}^A]$ matrix is diagonal wrt $\hat{e}_1, \hat{e}_2, \hat{e}_3$ csys, they are also the principal axes of inertia

Identifying p-axes at pt B (by visual inspection)

The e_3 -axis (z-axis) passes through point B.

Is the z-axis an axis of revolution for generating the composite body? NO

Are there identifiable planes of symmetry passing through B? YES



→ xz -plane → y -axis as p-axis of RB at B
→ yz -plane → x -axis as p-axis of RB at B

Since there are two orthogonal planes of symmetry, therefore, the intersecting axis of the planes of symmetry is also a p-axis ⇒ z-axis is also a p-axis (Lec 12)

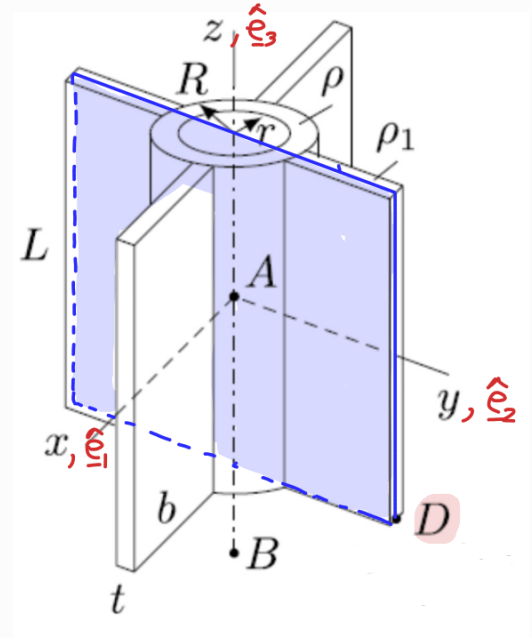
$$[\underline{I}^B] \begin{pmatrix} \hat{e}_1 \\ \hat{e}_2 \\ \hat{e}_3 \end{pmatrix} = \begin{bmatrix} I_{xx}^B & 0 & 0 \\ 0 & I_{yy}^B (= I_{zz}^B) & 0 \\ 0 & 0 & I_{zz}^B \end{bmatrix}$$

Identifying p-axes at pt D (by visual inspection)

Is any axis passing through D an axis of revolution for generating the composite body? NO

Are there identifiable planes of symmetry passing through D? YES

→ yz -plane → x -axis as p-axis
for RB at pt D

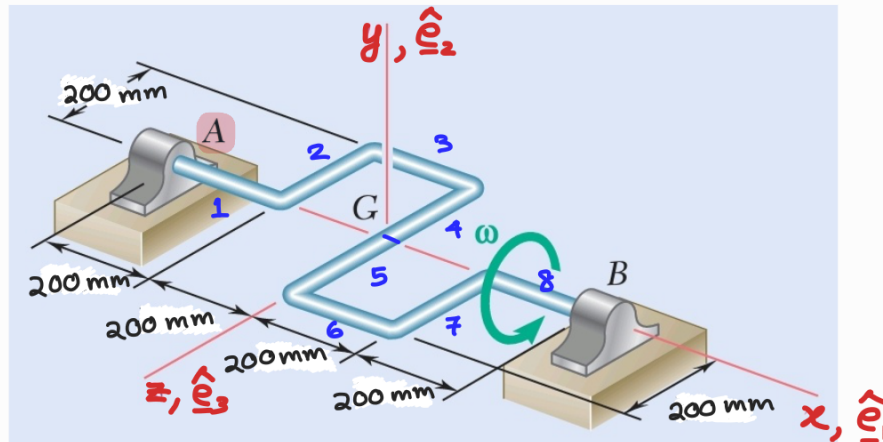


$$[\underline{\underline{I}}^D] \begin{pmatrix} \hat{e}_1 \\ \hat{e}_2 \\ \hat{e}_3 \end{pmatrix} = \begin{bmatrix} I_{xx}^D & 0 & 0 \\ 0 & I_{yy}^D & I_{yz}^D \\ 0 & I_{zy}^D & I_{zz}^D \end{bmatrix}$$

3) Find the angular momentum of the shaft about point A and w.r.t. ground with $\hat{e}_1 - \hat{e}_2 - \hat{e}_3$ as csys.

Mass of the shaft = 8 kg

Angular velocity = 12 rad/s



Mass of each shaft segment = $\frac{8 \text{ kg}}{8} = 1 \text{ kg}$

Solution:

$\underline{H}_{A|F}$: Angular momentum of the composite shaft having 8 simple rods

$\underline{\underline{I}}^A$: Inertia tensor of composite shaft

$\underline{\omega}_{m|F}$: Angular velocity of composite shaft

Given : $\underline{\omega}_{m|F} = \omega_1 \hat{e}_1$ (other components are zero)

$$\Rightarrow [\underline{\omega}_{m|F}] \begin{pmatrix} \hat{e}_1 \\ \hat{e}_2 \\ \hat{e}_3 \end{pmatrix} = \begin{bmatrix} \omega_1 \\ 0 \\ 0 \end{bmatrix}$$

$$[H_{A|F}] = \begin{bmatrix} I_{11}^A & I_{12}^A & I_{13}^A \\ I_{21}^A & I_{22}^A & I_{23}^A \\ I_{31}^A & I_{32}^A & I_{33}^A \end{bmatrix} \begin{bmatrix} \omega_1 \\ 0 \\ 0 \end{bmatrix}$$

only this col. matters since only $\omega_1 \neq 0$

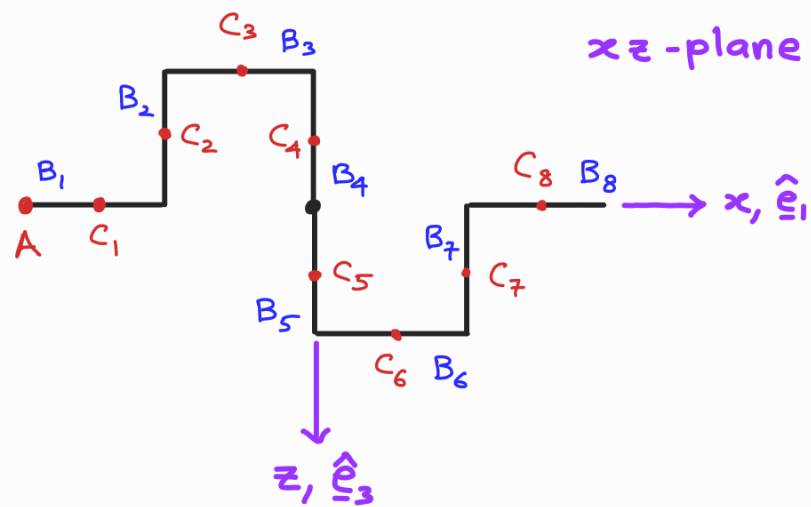
We need to find I_{11}^A , I_{21}^A , and I_{31}^A

Calculation of I_{21}^A

Since the shaft lies in the xz -plane, $I_{21}^A = 0$ (why?)

$$y \approx 0$$

$$I_{21}^A = - \int x y \, dm = 0$$

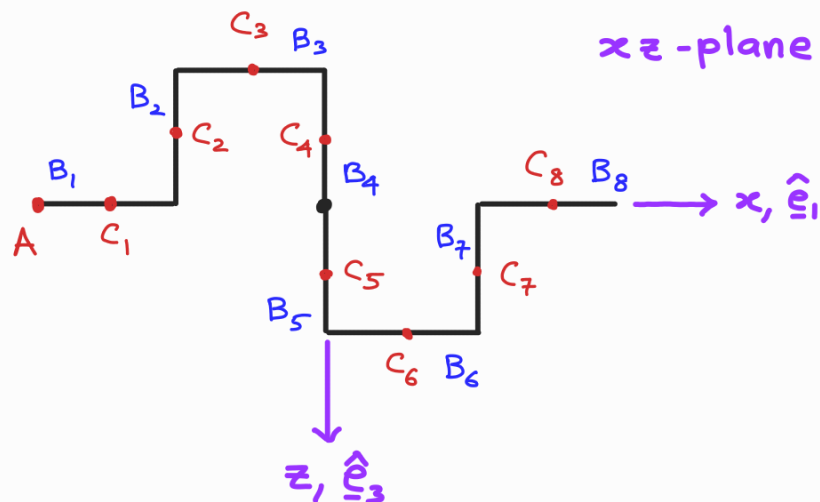


Calculation of I_{31}^A

We will first find I_{31}^C
(I_{31} at COM of each rod)
and then use // axes thm
to transfer to pt A.

Mass of each rod, $m_i = 1 \text{ kg}$

Length of each rod, $l_i = 0.2 \text{ m}$



Composite shaft $B = \bigcup_{i=1}^8 B_i$

$$\Rightarrow I_{31}^A = \sum_{i=1}^8 I_{31}^A(B_i)$$

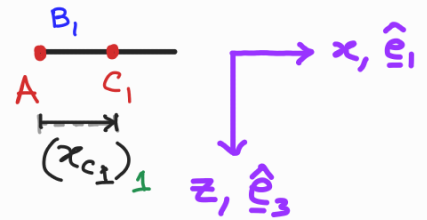
We will use the following relation:

$$I_{31}^A(B_i) = I_{31}^C(B_i) - m_i \overbrace{(x_{c_1})_3}^{\text{pos. from pt A to pt } C_i} \overbrace{(x_{c_4})_1}^{\perp \text{ to } \hat{e}_1}$$

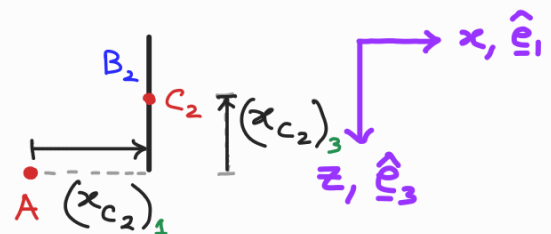
pos. from A to C_i
 $\perp$ to $\hat{e}_3$

Note that $I_{31}^C(B_i)$ is zero for all rods

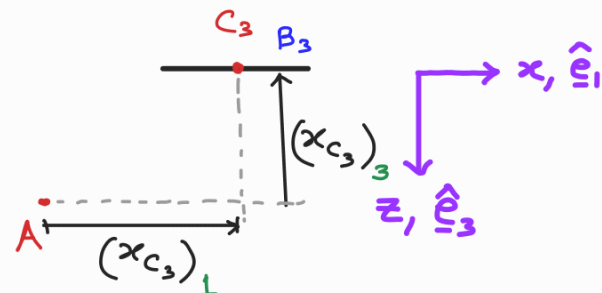
$$\begin{aligned} I_{31}^A(B_1) &= -m_1 \overbrace{(x_{c_1})_3}^0 (x_{c_1})_1 \\ &= 0 \end{aligned}$$



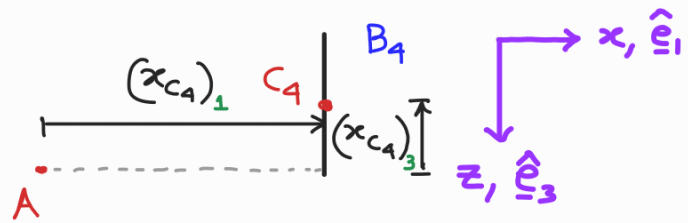
$$\begin{aligned} I_{31}^A(B_2) &= -m_2 (x_{c_2})_3 (x_{c_2})_1 \\ &= -m \left(-\frac{a}{2}\right) (a) \\ &= m \frac{a^2}{2} \end{aligned}$$



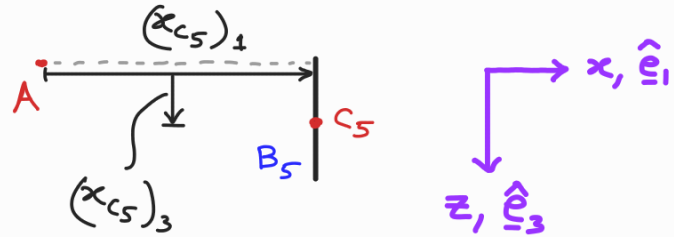
$$\begin{aligned} I_{31}^A(B_3) &= -m_3 (x_{c_3})_3 (x_{c_3})_1 \\ &= -m (-a) \left(a + \frac{a}{2}\right) \\ &= m \frac{3a^2}{2} \end{aligned}$$



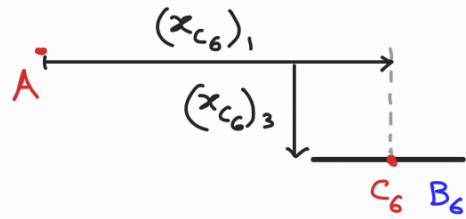
$$\begin{aligned}
 I_{31}^A(B_4) &= -m_4 (x_{c_4})_3 (x_{c_4})_1 \\
 &= -m \left(-\frac{a}{2}\right) (2a) \\
 &= m a^2
 \end{aligned}$$



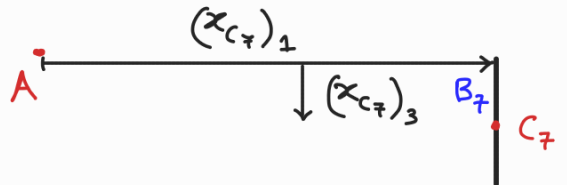
$$\begin{aligned}
 I_{31}^A(B_5) &= -m_5 (x_{c_5})_3 (x_{c_5})_1 \\
 &= -m \left(\frac{a}{2}\right) (2a) \\
 &= -m a^2
 \end{aligned}$$



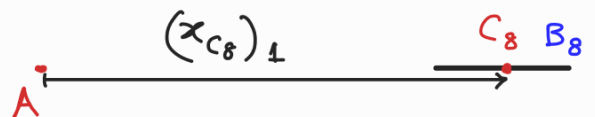
$$\begin{aligned}
 I_{31}^A(B_6) &= -m_6 (x_{c_6})_3 (x_{c_6})_1 \\
 &= -m (a) \left(2a + \frac{a}{2}\right) \\
 &= -m \left(\frac{5a^2}{2}\right)
 \end{aligned}$$



$$\begin{aligned}
 I_{31}^A(B_7) &= -m_7 (x_{c_7})_3 (x_{c_7})_1 \\
 &= -m \left(\frac{a}{2}\right) (3a) \\
 &= -m \left(\frac{3}{2} a^2\right)
 \end{aligned}$$



$$\begin{aligned}
 I_{31}^A(B_8) &= -m_8 (\cancel{x_{c_8}})_3 (\cancel{x_{c_8}})_1 \\
 &= 0
 \end{aligned}$$



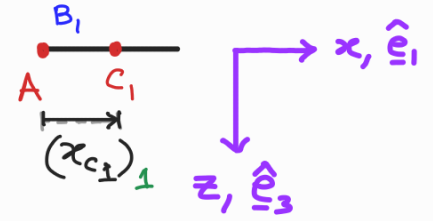
$$\begin{aligned}
 I_{31}^A(B) &= m \left[0 + \frac{a^2}{2} + \frac{\cancel{3a^2}}{2} + \cancel{a^2} - \cancel{a^2} - \frac{5a^2}{2} - \frac{\cancel{3a^2}}{2} \right] \\
 &= -2ma^2
 \end{aligned}$$

Calculation of I_{11}^A

$$I_{11}^A = \sum_{i=1}^8 I_{11}^A(B_i)$$

$$I_{11}^A(B_1) = I_{11}^{C_1}(B_1) + m[(x_{C_1})_2^2 + (x_{C_1})_3^2]$$

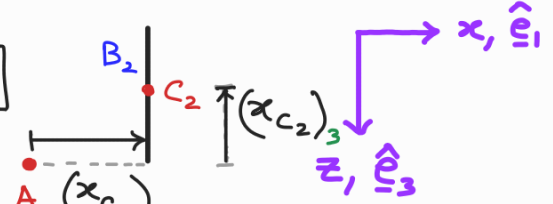
$$= 0$$



$$I_{11}^A(B_2) = I_{11}^{C_1}(B_2) + m[(x_{C_2})_2^2 + (x_{C_2})_3^2]$$

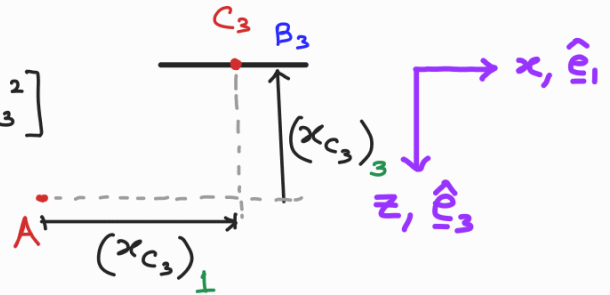
$$= \frac{ma^2}{12} + m\left[0 + \left(\frac{a}{2}\right)^2\right]$$

$$= \frac{ma^2}{3}$$



$$I_{11}^A(B_3) = I_{11}^{C_3}(B_3) + m[(x_{C_3})_2^2 + (x_{C_3})_3^2]$$

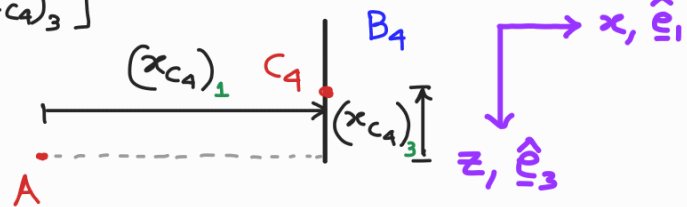
$$= ma^2$$



$$I_{11}^A(B_4) = I_{11}^{C_4}(B_4) + m[(x_{C_4})_2^2 + (x_{C_4})_3^2]$$

$$= \frac{ma^2}{12} + m\left(\frac{a}{2}\right)^2$$

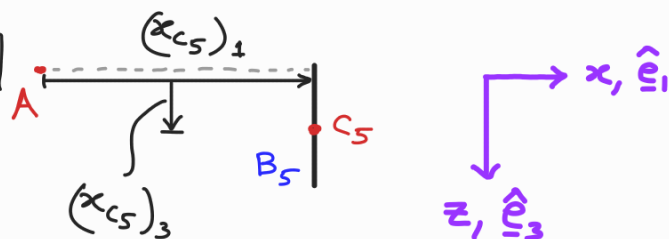
$$= \frac{ma^2}{3}$$



$$I_{11}^A(B_5) = I_{11}^{C_5}(B_5) + m[(x_{C_5})_2^2 + (x_{C_5})_3^2]$$

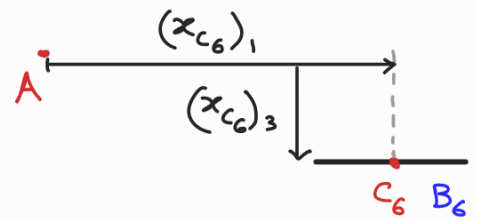
$$= \frac{ma^2}{12} + m\left(\frac{a}{2}\right)^2$$

$$= \frac{ma^2}{3}$$



$$I_{11}^A(B_6) = I_{11}^{C_6}(B_6) + m[(x_{C_6})_2^2 + (x_{C_6})_3^2]$$

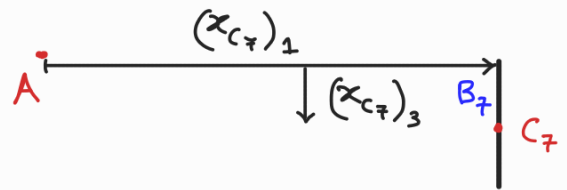
$$= ma^2$$



$$I_{11}^A(B_7) = I_{11}^{C_7}(B_7) + m[(x_{C_7})_2^2 + (x_{C_7})_3^2]$$

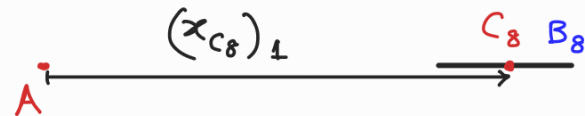
$$= \frac{ma^2}{12} + m\left(\frac{a}{2}\right)^2$$

$$= \frac{ma^2}{3}$$



$$I_{11}^A(B_8) = I_{11}^{C_8}(B_8) + m[(x_{C_8})_2^2 + (x_{C_8})_3^2]$$

$$= 0$$



$$I_{11}^A(B) = \sum_{i=1}^8 I_{11}^A(B_i)$$

$$= 0 + \frac{ma^2}{3} + ma^2 + \frac{ma^2}{3} + \frac{ma^2}{3} + ma^2 + \frac{ma^2}{3} + 0$$

$$= 2ma^2 + \frac{4}{3}ma^2$$

$$= \frac{10}{3}ma^2$$

Therefore, $[H_{AIF}] \begin{pmatrix} \hat{e}_1 \\ \hat{e}_2 \\ \hat{e}_3 \end{pmatrix} = ma^2 \omega \begin{bmatrix} 10/3 \\ 0 \\ -2 \end{bmatrix}$